

Working life calculation

The nominal theoretical working life of a ball screw is calculated by a method similar to that for calculating the working life of a ball bearing. It should be noted that vibration and shock loads adversely affect the working life of the ball screw. Radial loadings are not permitted.

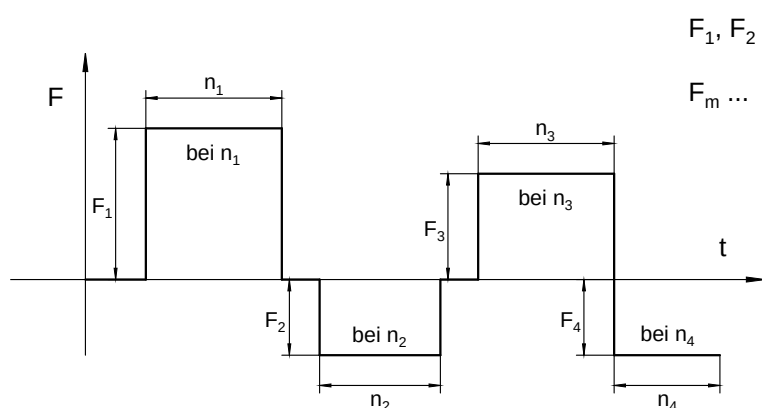
Average speed:

$$n_m = \frac{n_1 \cdot q_1 + n_2 \cdot q_2 + \dots + n_i \cdot q_i}{100}$$

$n_m \dots$ [min⁻¹] average speed
 $n_1, n_2 \dots$ [min⁻¹] speeds during the interval $q_1, q_2, \dots$
 $q_1, q_2 \dots$ [%] proportions of the duration of loaded operation in one direction of loading

Dynamic equivalent axial load:

$$F_m = \sqrt[3]{F_1^3 \cdot \frac{n_1 \cdot q_1}{n_m \cdot 100} + F_2^3 \cdot \frac{n_2 \cdot q_2}{n_m \cdot 100} + \dots + F_i^3 \cdot \frac{n_i \cdot q_i}{n_m \cdot 100}}$$



$F_1, F_2 \dots$ [N] Axial loads in one direction of loading

$F_m \dots$ [N] Dynamic equivalent axial load
 Since a ball screw can be loaded in either of two directions, F_m is first determined for each of the two directions of loading. The larger value is then used in the calculation of L . In general it is useful to create the following structure:

It should be remembered that a pre-load represents an ever-present additional load.

Theoretical working life:

$$L_{10} = \left(\frac{C}{F_m} \right)^3 \cdot 10^6$$

C [N] Dynamic load rating
 Centrally applied load of unchangeable value and direction, for which a sufficiently large number of identical ball screws each achieves a nominal working life of 10^6 revolutions.

L_{10} Working life of the ball screw
 Expressed as the number of overrollings which are reached or exceeded by 90 % of a sufficiently large number of apparently identical ball screws before the first signs of material fatigue appear.

(Working life in metres: L_{10} multiplied by the pitch, divided by 1000)