Deflection belt drive (DBD) pretension Version: Standard



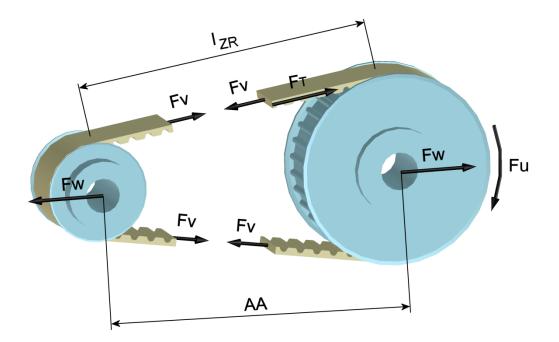
General

The purpose of pretension is to guarantee a minimum tension force in the slack strand in order to ensure smooth engagement in the driven disc. The set pretension should generally only be as high as necessary due to the fact that it acts as an additional load on the bearings. In this regard, the necessary pretension force F_V of the belt strands is dependent on the max. circumferential force F_U , the belt length L_B (or number of teeth Z_B) and the drive configuration. In the case of the belts in our deflection belt drives, a pretension force of $F_V = 1/2 F_U$ is recommended by the manufacturer. The cable winch strength is the upper limit for the strand load in all cases.

The pretension force is measured using the frequency measurement performed on a 'rubbed' belt strand using a belt tension measuring device.

Cable winch strength and belt mass of our standard DBD versions:

- DBD1: 1260 N / 0.054 kg/m (16AT5) => Example: Belt width 16 mm, toothing AT5
- DBD2: 2030 N / 0.085 kg/m (25AT5)
- DBD3: 2660 N / 0.109 kg/m (32AT5) 4750 N / 0.202 kg/m (32AT10)



Centre distance	AA	[mm]	Pretension force	F∨	
Strand force	F _T	[N]	Shaft force	Fw	
Circumferential force	e Fu	[N]	Toothed belt length (oscillating)	I_{ZR}	[mm]

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Calculation

AA	: Centre distance	[mm]
d_{0A}	: Effective diameter	[mm]
	(of output synchronised pulle	ey)
d _{om}	: Effective diameter	[mm]
	(of motor synchronised pulle	ey)
f_{ZR}	: Pretension frequency	[Hz]
F_{T}	: Strand force	[N]
Fυ	: Circumferential force (= F_T)	[N]
F _v	: Pretension force	[N]
F_{zul}	: Cable winch strength	[N]

I _{gesZR}	: Toothed belt total length	[mm]
I_{ZR}	: Toothed belt length	[mm]
	(oscillating)	
m _{zR}	: Toothed belt mass	[kg/m]
M _{Lin}	: Drive moment	[Nm]
	on linear unit	
t _{zR}	: Belt tooth pitch	[mm]
Z _A	: Number of teeth	[]
	(on output synchronised p	ulley)
ZM	: Number of teeth	[]
	(on motor synchronised pu	ılley)

Туре	DBD1 / AT5				DBD2 / AT5					
i	1:1	1.5:1	2:1	3:1	1:1	1:1	1.5:1	2:1	3:1	
t _{zr}	5				5					
I _{gesZR}	390	375	390	375	420	390	420	480	450	
F_{zul}	1260					2030				
m _{zR}	0.054				0.085					
Z _A	36	36	48	48	36	30	36	60	60	
z _M	36	24	24	16	36	30	24	30	20	
AA	105	112	103	104	120	120	119	125	120	

Туре		DBD3	/ AT5	DBD3 / AT10			
i	1:1	1.5:1	2:1	3:1	1:1	1.5:1	2:1
t _{zR}		Ę	5	10			
I _{gesZR}	600	630	630	630	610	630	630
F_{zul}		26	60	4750			
m _{ZR}		0.1	09	0.202			
Z _A	44	60	72	72	22	30	36
Z _M	44	40	36	24	22	20	18
AA	190	189	177	191	195	189	177

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Calculation/formulas

Determination of the circumferential force ${\rm F}_{\rm U}$

$$F_U = \frac{2 \cdot M_{\text{Lin}}}{d_{0A}}$$
 with $d_{0A} := \frac{z_A \cdot t_{ZR}}{\pi}$

(F $_{\rm U}$ must be <= cable winch strength F $_{\rm zul}$ / 1.5.)

Calculation of pretension force ${\sf F}_{\sf V}$

$$F_V := F_U \cdot \frac{1}{2}$$

Calculation of oscillating belt length I_{ZR}

$$I_{ZR} := \sqrt{AA^2 - \left(\frac{d_{0A} - d_{0M}}{2}\right)^2} \quad \text{with} \quad d_{0A} := \frac{z_A \cdot t_{ZR}}{\pi} \quad \text{and} \quad d_{0M} := \frac{z_M \cdot t_{ZR}}{\pi}$$

Calculation of pretension frequency to be configured $\ensuremath{\mathsf{f}_{\mathsf{ZR}}}$

$$f_{ZR} = \sqrt{\frac{F_V}{4 \cdot m_{ZR} \cdot l_{ZR}^2}}$$

In order to measure the pretension frequency f_{ZR} get the belt strand oscillating ("rubbing") using a suitable tool (e.g. Allen key) and use a belt tension measuring device around the centre of the strand to determine the frequency.

(When measuring the frequency, the fastening screws for the motor must be tightened.)