# Deflection belt drive pretension (DBD) Version: standard



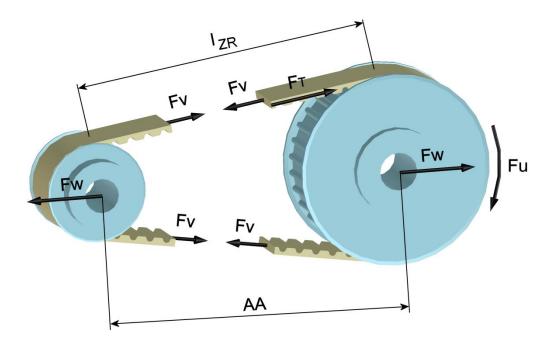
#### General

Pretension is tasked with guaranteeing a minimum tension force in the slack strand in order to ensure smooth engagement in the driven disc. The configuration of the pretension should generally only be as large as necessary due to the fact that it acts as an additional load on the bearings. In this regard, the necessary pretension force  $F_V$  of the belt strands is dependent on the max. circumferential force  $F_U$ , the belt length  $L_B$  (or number of teeth  $Z_B$ ) and the drive configuration. In the case of the belts in our deflection belt drives, a pretension force of  $F_V = 1/2$   $F_U$  is recommended by the manufacturer. The cable winch strength is the upper limit for the strand load in all cases.

The pretension force is measured via the frequency measurement performed on a 'beating' belt strand using a belt tensioning measurement device.

#### Cable winch strength and belt mass of our DBD standard versions:

- DBD1: 1260 N / 0.054 kg/m (16AT5) => Example: belt width 16 mm, toothing AT5
- DBD2: 2030 N / 0.085 kg/m (25AT5)
- DBD3: 2660 N / 0.109 kg/m (32AT5) 4750 N / 0.202 kg/m (32AT10)



Centre distance	AA	[mm]	Pretension force	Fv	[N]
Strand force	$F_T$	[N]	Shaft force Toothed belt length	Fw I <sub>zr</sub>	[N] [mm]
Circumferential force	e Fu	[N]	(oscillating)	'ZR	[]



## Calculation

AA	: Centre distance	[mm]
d <sub>0A</sub>	: Effective diameter	[mm]
	(of synchronising disc, driven)	
d <sub>om</sub>	: Effective diameter	[mm]
	(of synchronising disc, motor)	
$f_{ZR}$	: Pretension frequency	[Hz]
FT	: Strand force	[N]
Fυ	: Circumferential force (= $F_T$ )	[N]
F <sub>v</sub>	: Pretension force	[N]
$F_{zul}$	: Cable winch strength	[N]

I <sub>gesZR</sub>	: Toothed belt total length	[mm]
I <sub>ZR</sub>	: Toothed belt length	[mm]
	(oscillating)	
m <sub>zr</sub>	: Toothed belt mass	[kg/m]
$M_{Lin}$	: Drive torque	[Nm]
	on linear unit	
t <sub>zR</sub>	: Belt tooth pitch	[mm]
Z <sub>A</sub>	: Number of teeth	[]
	(on synchronising disc, driv	/en)
z <sub>M</sub>	: Number of teeth	[]
	(on synchronising disc, mo	tor)

Туре	DBD1 / AT5				DBD2 / AT5					
i	1:1	1.5:1	2:1	3:1	1:1	1:1	1.5:1	2:1	3:1	
t <sub>ZR</sub>	5				5					
I <sub>gesZR</sub>	390	375	390	375	420	390	420	480	450	
$F_{zul}$	1260					2030				
m <sub>ZR</sub>	0.054					0.085				
Z <sub>A</sub>	36	36	48	48	36	30	36	60	60	
z <sub>M</sub>	36	24	24	16	36	30	24	30	20	
AA	105	112	103	104	120	120	119	125	120	

Туре		DBD3	/ AT5	DBD3 / AT10			
i	1:1	1.5:1	2:1	3:1	1:1	1.5:1	2:1
t <sub>ZR</sub>		Ę	5	10			
I <sub>gesZR</sub>	600	630	630	630	610	630	630
$F_{zul}$		26	60	4750			
m <sub>ZR</sub>		0.1	09	0.202			
Z <sub>A</sub>	44	60	72	72	22	30	36
z <sub>M</sub>	44	40	36	24	22	20	18
AA	190	189	177	191	195	189	177

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## Calculation/formulas

Determination of the circumferential force  ${\rm F}_{\rm U}$ 

$$F_U = \frac{2 \cdot M_{\text{Lin}}}{d_{0A}}$$
 with  $d_{0A} := \frac{z_A \cdot t_{ZR}}{\pi}$ 

(  $\rm F_{U}$  must be <= cable winch strength  $\rm F_{zul}$  / 1.5. )

Calculation of pretension force  ${\sf F}_{\sf V}$ 

$$F_V := F_U \cdot \frac{1}{2}$$

Calculation of oscillating belt length IZR

$$I_{ZR} := \sqrt{AA^2 - \left(\frac{d_{0A} - d_{0M}}{2}\right)^2} \quad \text{with} \quad d_{0A} := \frac{z_A \cdot t_{ZR}}{\pi} \quad \text{and} \quad d_{0M} := \frac{z_M \cdot t_{ZR}}{\pi}$$

Calculation of pretension frequency to be configured  $f_{ZR}$ 

$$f_{ZR} = \sqrt{\frac{F_V}{4 \cdot m_{ZR} \cdot l_{ZR}^2}}$$

In order to measure the pretension frequency  $f_{ZR}$ , use a suitable tool (e.g. Allen key) to make the belt strand oscillate ('beat') and use a belt tensioning measurement device around the centre of the strand to determine the frequency.

(When measuring the frequency, the fastening screws of the motor must be tightened.)